

$OAGH$ is the acceleration diagram.

$$f_{ba}^n = \omega^2 \cdot AG$$

$$f_{ba}^t = \omega^2 \cdot GH$$

$$f_{ba} = \omega^2 \cdot AH$$

$$\omega^2 = \left(\frac{2\pi \times 300}{60} \right)^2 = 986.96 \text{ (rad/s)}^2$$

Locate point X in AB , such that $AX = 400$ mm. Draw Xx parallel to BO to meet AH at x . Join Ox . Then

$$Ox \propto f_x$$

$$f_b = \omega^2 OH$$

$$(a) \text{ Acceleration of piston, } f_b = \omega^2 \times OH = \frac{986.96 \times 1.1 \times 100}{1000} = 108.56 \text{ m/s}^2$$

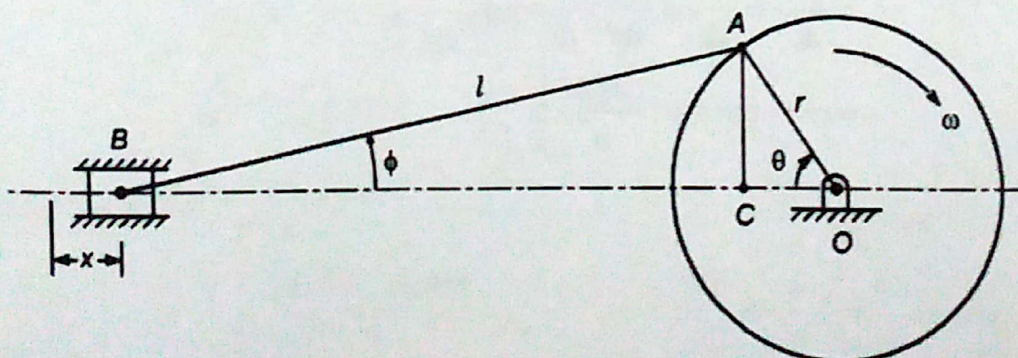
$$(b) \text{ Tangential acceleration of rod, } f_{ba}^t = \omega^2 \times GH = \frac{986.96 \times 2.6 \times 100}{1000} = 256.61 \text{ m/s}^2$$

$$\text{Angular acceleration of rod, } \alpha_{AB} = \frac{f_{ba}^t}{AB} = \frac{256.61}{1.2} = 213.84 \text{ rad/s}^2$$

$$(c) \text{ Acceleration of point } X \text{ on rod} = \omega^2 \times Ox = \frac{986.96 \times 2.2 \times 100}{1000} = 217.13 \text{ m/s}^2$$

3.7 ANALYTICAL ANALYSIS OF SLIDER-CRANK MECHANISM

Consider the slider-crank mechanism shown in Fig.3.26. Let θ be the angle turned through by the crank $OA = r$ when the slider B has moved by an amount x to the right, and ϕ the angle, which the connecting rod $AB = l$ makes with the line of stroke.



$$\begin{aligned}x &= r + l - (OC + BC) \\ &= r + l - (r \cos \theta + l \cos \phi) \\ &= r(1 - \cos \theta) + l(1 - \cos \phi)\end{aligned}$$

Now

$$AC = r \sin \theta = l \sin \phi$$

$$\sin \phi = \frac{r}{l} \sin \theta$$

Let

$$n = \frac{l}{r}$$

Then

$$\cos \phi = (1 - \sin^2 \phi)^{0.5}$$

$$= \left[1 - \frac{\sin^2 \theta}{n^2} \right]^{0.5}$$

$$x = r(1 - \cos \theta) + l \left\{ 1 - \left(1 - \frac{\sin^2 \theta}{n^2} \right)^{0.5} \right\}$$

$$= r \left[(1 - \cos \theta) + n \left\{ 1 - \left(1 - \frac{\sin^2 \theta}{n^2} \right)^{0.5} \right\} \right]$$

$$= r \left[(1 - \cos \theta) + n \left\{ 1 - \left(1 - \frac{\sin^2 \theta}{2n^2} - \dots \right) \right\} \right]$$

$$\approx r \left[(1 - \cos \theta) + \frac{\sin^2 \theta}{2n} \right]$$

$$\approx r \left[(1 - \cos \theta) + \frac{(1 - \cos 2\theta)}{4n} \right]$$

(3.13)

Velocity of slider, $v_b = \frac{dx}{dt} = \frac{dx}{d\theta} \cdot \frac{d\theta}{dt} = \frac{dx}{d\theta} \cdot \omega$

$$= \omega r \left[\sin \theta + \frac{\sin 2\theta}{2n} \right]$$

(3.14)

Acceleration of slider,

$$f_b = \frac{d^2x}{dt^2} = \frac{dv_b}{dt} = \frac{dv_b}{d\theta} \cdot \frac{d\theta}{dt} = \omega \cdot \frac{dv_b}{d\theta}$$

$$= \omega^2 r \left[\cos \theta + \frac{\cos 2\theta}{n} \right]$$

Now

$$\sin \phi = \frac{\sin \theta}{n}$$

$$\begin{aligned}\cos \phi \cdot \frac{d\phi}{dt} &= \frac{\cos \theta}{n} \times \frac{d\theta}{dt} \\ &= \frac{\omega \cos \theta}{n}\end{aligned}$$

Angular velocity of connecting rod,

$$\begin{aligned}\omega_{ba} &= \frac{d\phi}{dt} = \frac{\omega}{n} \cdot \frac{\cos\theta}{\cos\phi} \\ &= \frac{\omega \cos\theta}{(n^2 - \sin^2\theta)^{0.5}} \\ &\approx \left(\frac{\omega}{n}\right) \cos\theta\end{aligned}\quad (3.15)$$

Angular acceleration of connecting rod,

$$\begin{aligned}\alpha_{ba} &= \frac{d\omega_{ba}}{dt} = -\omega^2 \sin\theta \frac{(n^2 - 1)}{(n^2 - \sin^2\theta)^{3/2}} \\ &\approx -\frac{\omega^2}{n} \cdot \sin\theta\end{aligned}\quad (3.16)$$

Example 3.15